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# Simulation and control of three-dimensional wakes

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## Abstract

7 We illustrate the problem of wake optimization on two examples. In the first example, the destruction of  
8 trailing vortices, we show that stochastic optimization is a flexible tool to identify vortex systems enhancing  
9 natural instabilities. In the second example, the problem of drag reduction in a cylinder wake, a priori  
10 information on the flow dynamics is highly desirable and we give some preliminary results which should be  
11 useful to select the most efficient parameters in an optimization strategy based on differential rotation of the  
12 cylinder.

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## 15 1. Introduction

16 The control of wakes is a subject of paramount importance in aircraft and automobile industry.  
17 Depending on the particular application, wake control can have various goals and can be  
18 achieved either by passive or active strategies. Passive control mostly operates through shape  
19 optimization and often results in the addition of appendices like foilers or ribbllets to the surface of  
20 the obstacle. Active control implies that one is ready to impart energy on the flow by means of  
21 actuators on the surface of the obstacle, keeping in mind that this energy must be included in the  
22 global energy budget to conclude on the efficiency of the particular control strategy.

23 We focus here on two examples, each of them emphasizing a specific challenge of active control  
24 in 3D wakes. The first example is the destruction of trailing vortices shed by airplane wings. The  
25 second example is the drag reduction behind a bluff body. Both examples illustrate the need of  
26 active control. In the first case, there is little hope that any reasonable passive device could prevent  
27 the formation, or significantly modify the characteristics, of tip vortices that are created by the

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28 wings and flaps of an aircraft. In the second case, the automobile industry is a good example  
29 where shape optimization has led to important improvements in the last decades, but nowadays  
30 shows its limits, mostly due to design considerations. In the coming years, new drastic regulations  
31 in pollutants emissions will impose to explore new directions and in particular active control  
32 strategies.

33 These strategies, beside the technology issues that they will raise, will be very demanding in  
34 terms of simulation and optimization tools. Three-dimensional wakes are still a very challenging  
35 field for simulation methods, because of the complex unsteady features of the flows. Concerning  
36 optimization itself, if optimal control has been able to give interesting results for 2D flows [9,17],  
37 there is still a long way to a systematic approach of 3D flows. Close loop control, where actuation  
38 would be imposed adaptively in terms of sensor informations is an even more open field, with the  
39 noticeable exception of [10] where it is shown that pressure measurements on the surface of the  
40 obstacle can be turned into vorticity fluxes and, ultimately, blowing and suction to significantly  
41 reduce the drag. Most of the current works in the field still aim at gaining insights into the dy-  
42 namics of the flows in order to propose simple open loop strategies.

43 In this paper we review some recent or on-going efforts to implement evolution strategies for the  
44 optimization of 3D wakes. Evolution strategies are stochastic optimization methods where pa-  
45 rameters are optimized through random search followed by selection on the basis of the evalu-  
46 ation of an objective function. Their convergence properties are not as well established as for  
47 gradient based methods, but they do not require to differentiate objective functions and are thus  
48 rather straightforward to implement. Other advantages are their ability to track global, instead of  
49 local, optima, their natural parallelism and their flexibility in accounting for constraints or a priori  
50 informations in the parameter space. The trade-off is clearly that they are CPU time consuming, in  
51 particular for the optimization of 3D unsteady flows, because their convergence is slow and every  
52 iteration involves a complete run. Evolution strategies have already proved to be very efficient for  
53 several instances of flow optimization. We refer to [19] and the references therein for examples in  
54 mixing and jet optimization.

55 In the sequel, we first show in Section 2 that the destruction of trailing vortices is a striking case  
56 where evolution strategy is successful. The starting point here was a study by Crouch [5] of co-  
57 operative instabilities of two pairs of vortex tubes. In that study some particular parameter values  
58 where shown to potentially enhance vortex break-up, on a time scale much smaller than classical  
59 single-pair Crow instabilities. In this example, the problem setting is rather simple and simulations  
60 are not very expensive. Evolution strategies not only could rapidly converge to a solution very  
61 close to the particular parameters exhibited in [5], but also could open up to other, more efficient,  
62 solutions.

63 In Section 3, we turn to the problem of drag reduction in a cylinder wake. In that case, in order  
64 to derive successful evolution strategies the complexity of the dynamics makes it crucial to reduce  
65 the parameter space dimension. One way to achieve this dimension reduction is to inject a priori  
66 informations available for this type of flows. Section 3 is devoted to that particular problem for  
67 the cylinder wake. We in particular investigate the links between three-dimensional features in the  
68 wake and drag values. We first consider two-dimensional control strategies, based on cylinder  
69 rotations then on 2D differential rotation parameters identified in [11]. We show that these control  
70 strategies on 3D flows make the flow return to a 2D state. Based on the 3D instability modes that

71 naturally develop in uncontrolled 3D wakes, we then explore some 3D control profiles and discuss  
 72 their potential efficiency in the context of drag optimization.

73 **2. Optimization of trailing vortices**

74 The control of trailing vortices remains a challenging problem of strong economical interest.  
 75 These vortices are naturally shed by airplanes and cause strong down-wash that are a hazard for  
 76 following aircrafts. Many studies have focused on the so-called Crow instability that spontane-  
 77 ously break-up these vortices [20]. Recently some studies [5,18] have considered cooperative in-  
 78 stabilities resulting from the interaction of several vortex pairs. The starting point of the study in  
 79 [5] is the identification of a complex system of vortices shed by the wings, the flaps and the fuselage  
 80 of a typical aircraft. This vortex system is sketched in Fig. 1. The two pairs originating from the  
 81 wing tips and the inboard flaps are of particular interest. Crouch [5] shows that, depending on the  
 82 perturbations that are imparted to these two pairs of initially parallel tubes, several instability  
 83 regimes can be observed, respectively termed long-wave, transient growth, and short-wave in-  
 84 stabilities (Fig. 1, right picture). In particular, the transient growth regime was shown to produce  
 85 a growth rate far exceeding that of a single pair. Based on these findings Crouch and Spalart  
 86 proposed to implement active control devices to trigger the proper instabilities. Our goal in this  
 87 study was to identify the fastest growing instability by performing a systematic search in the  
 88 parameter space defining the initial state of the perturbed vortex pair, having in mind that the

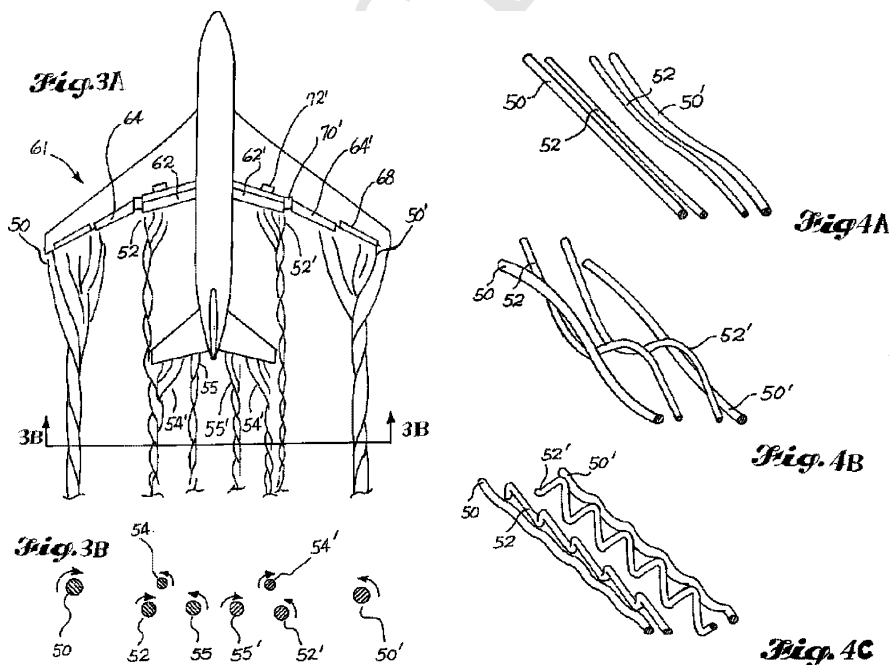


Fig. 1. Sketch of vortex system shed by an airplane (left picture) and of long-wave, transient growth and short-wave instabilities (right picture, top to bottom) (Courtesy of J. Crouch).

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89 success of such an approach could open up to the investigation of a broader class of vortex  
90 systems with the possibility of further accelerating the break-up process.

### 91 2.1. The evolution strategy

92 The principle of evolution strategies (ES) is an exploration of the parameter space by a random  
93 walk followed by a simple selection process. If  $f$  is the function to minimize and  $X \in R^N$  denotes  
94 the parameter vector, an iteration of a so-called one-member strategy can be summarized by

$$X_{t+1} = \begin{cases} X_t + \sigma_t Z_t & \text{if } f(X_t + \sigma_t Z_t) \leq f(X_t) \\ X_t & \text{otherwise} \end{cases}$$

96 In the above formula,  $Z_t$  denotes a random Gaussian vector with zero mean and unit standard  
97 deviation. The radius  $\sigma_t$  of the random walk is updated in function of the success rate at the  
98 previous iterations in order to optimize the convergence speed. A high success rate meaning that  
99 one is far away from the minimum induces an increase in  $\sigma_t$ . In this work we have implemented  
100 the so-called 1/5 rule: the variance is increased if the success ratio during the last iterations is  
101 greater than 1/5.

102 The method just described is isotropic in the sense that the random walk is done in all directions  
103 with equal probability. Convergence speed-up can be reached if one adapts the random walk in  
104 the direction of best success rate. This is the so-called covariance matrix adaptation (CMA)  
105 technique (see [7]). Instead of mutating a single parameter (one-member strategy) one may also  
106 consider mutations and recombinations among several individuals (multi-member strategies). For  
107 the present purpose, the plain one-member ES proved to be sufficient.

### 108 2.2. Results

109 Our study focused essentially on the case of two pairs of co-rotating vortices in the configu-  
110 ration studied by Crouch. The parameters on which the evolution strategies optimized were:

- 111 • the perturbation initial amplitude of the tip ( $\epsilon_1$ ) and outboard ( $\epsilon_2$ ) vortices,
- 112 • the angles of the perturbation planes  $\alpha_1, \alpha_2$ ,
- 113 • the wavelength of the perturbations  $\lambda$ ,
- 114 • the separation between the two vortices  $\delta$ ,
- 115 • the circulation ratio between the outboard and tip vortices  $\Gamma$ .

116 Quantities were adimensionalized by the span (distance between the vorticity centers of each  
117 pair) which was always kept constant, and the total circulation of each pair. The computations  
118 were done on a periodic box of size  $2 \times 2 \times \lambda$  by a vortex method using  $64^3$  elements (see Section  
119 3.1 for details on the numerics).

120 To work with parameters in the same order of magnitude as in [5], the total perturbation was  
121 constrained to be below 10% of the span:

$$\epsilon_1^2 + \epsilon_2^2 \leq 0.01$$

123 The following additional constraints were imposed on these parameters to remain within  
124 achievable design configurations:

$$0.25 \leq \delta \leq 0.4; \quad 0.5 \leq \lambda \leq 10; \quad 0.0 \leq \Gamma \leq 0.5$$

126 Note that the constraints on  $\lambda$  allow for a wide range of wavelengths, varying from short  
127 wavelength of the order of a few core sizes, to long wavelengths of the type found in Crow in-  
128 stability.

129 Our goal was to optimize the instability on the tip, stronger, vortex. To measure the defor-  
130 mation of this vortex, we computed the average angle, inside the core of the tip vortex, of the  
131 vorticity vector  $\omega$  with the base flow axis. More precisely the objective function was given by the  
132 formula

$$f = \int dz \int_{A(z)} \frac{\omega_x^2 + \omega_y^2}{\omega_z^2} dA(z)$$

134 where

$$A(z) = \{(x, y), |\omega(x, y, z)| \geq \frac{1}{2} |\omega|_{\max}\}$$

136 Fig. 2 shows the convergence history of the evolution algorithm.

137 The parameter values finally obtained by the ES are listed in Table 1, together with the pa-  
138 rameters that are reported in [5] to lead to efficient transient growth. Some striking similarities can  
139 be noticed between these two sets of parameters. In particular the ES has selected perturbations

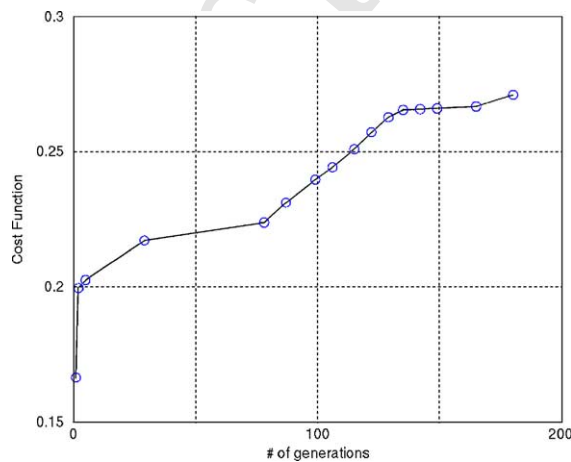


Fig. 2. Convergence history for the evolution strategy.

Table 1

Comparison of the parameters found by the evolution strategy and those studied in [5]

	$\alpha_1$	$\alpha_2$	$\epsilon_1$	$\epsilon_2$	$\delta$	$\Gamma$	$\lambda$
Optimal parameters	0.47	0.73	0.098	0.008	0.26	0.31	0.72
Parameters in [5]	$\pi/4$	$\pi/4$	0.1	0.0	0.3	0.4	0.7

140 that are mostly located on the tip vortex ( $\epsilon_2 \ll \epsilon_1$ ), confirming the observation in [5] of efficient  
141 transient growth when the outboard vortex was unperturbed. The wavelength of the perturbations  
142 ( $\lambda$ ) is also very close to the one given in [5].

### 143 2.3. Perspectives and open questions

144 Trailing vortices are a good example of an unsteady complex flow where evolution strategies are  
145 successful to retrieve optimal parameters. One reason is that the simplicity of the flow geometry  
146 enables simulations which are not too expensive. For that particular example, the flexibility of this  
147 optimization technique should allow to investigate more complex vortex systems. We actually ran  
148 the evolution strategy on 4 pairs, 2 of which being of opposite circulation, playing the role of  
149 vortices 54 and 55 in Fig. 1. We did obtain parameter values for this system which even improved  
150 the optimal growth rate for two pairs. However one open and critical question that remains to be  
151 solved is the time span on which the objective should be measured for this particular application.  
152 We did observe that, for 4 as well as for 2 pairs, the growth rates significantly decreased after the  
153 control time. This confirms observation made in [18] and there is some doubt that these pa-  
154 rameters are indeed optimal for fastest reconnection. Moreover the growth rates proved to be  
155 rather sensitive to parameter values. The present findings should thus be taken with caution.  
156 Nevertheless we believe that the ease evolution strategies have to incorporate constraints of any  
157 kind in the parameter range, in particular to fit current design constraints, make them an ap-  
158 pealing tool for further studies.

### 159 3. Control of three-dimensional wake behind a cylinder

160 One important drawback of evolution strategies for unsteady flows is their computational cost.  
161 If, as the above example demonstrated, one-member strategies can be efficient for 3D flows in  
162 simple geometries, for more complex problems like drag optimization in bluff-body flows parallel  
163 implementations of multi-member strategies are certainly desirable. Moreover, acceptable con-  
164 vergence speed requires to work on parameter spaces as small as possible. For this purpose, it is  
165 crucial to accumulate as much knowledge as possible on the flow dynamics in order to select the  
166 appropriate parameters to optimize.

167 We show in the sequel some preliminary results along these lines for the case of a flow past a  
168 cylinder. The choice of the cylinder is dictated by the fact that it embodies many generic features  
169 of 3D dynamics and that several experimental and numerical results exist to validate the proposed  
170 methods. Concerning the particular problem of drag optimization, some results exist in 2D. In  
171 particular, optimal control techniques have been applied to propose cylinder rotation or suction  
172 and blowing parameters to suppress the shedding [8,9] or reduce the drag [17]. However to our  
173 knowledge these methods have not been extended to 3D flows. Stochastic drag optimization has  
174 also been recently used for 2D cylinder [11], and our on-going efforts aim at extending these  
175 results to the 3D case.

176 Whatever control technique is used, applications to 3D flows are always very demanding on the  
177 numerical side. Suction, blowing and body rotations dramatically change the boundary layer  
178 dynamics by introducing strong dipoles in the flow. The particle solver that we have used seems to

179 meet the robustness requirements to handle these features. In the sequel we first sketch the nu-  
180 merical code and then discuss some 3D control strategies that give some interesting directions for  
181 future optimization.

### 182 3.1. A particle-in-cell method for 3D bluff-body flows

183 We consider the velocity–vorticity form of the incompressible Navier–Stokes equations:

$$\frac{\partial \boldsymbol{\omega}}{\partial t} + (\mathbf{u} \cdot \nabla) \boldsymbol{\omega} - (\boldsymbol{\omega} \cdot \nabla) \mathbf{u} - \nu \Delta \boldsymbol{\omega} = 0 \quad (1)$$

185 Particle-in-cell methods combine particles and grids to discretize these equations: particles carry  
186 vorticity and implicitly take in account transport terms in the equation, while velocity and strain  
187 are computed on an underlying grid. The no-slip boundary condition is enforced through vorticity  
188 flux formulas.

189 Details of these algorithms can be found in [2–4,12–14]. Refs. [4,14,15] more specifically deal  
190 with design, validation and results in cylindrical geometry. For a sake of completeness we outline  
191 here the main features of the algorithm. Each particle, with index  $p$ , carries an element of vorticity  
192  $\boldsymbol{\omega}_p$ , with volume  $v_p$  and location  $\mathbf{x}_p$ . These quantities satisfy the system of differential equations:

$$\frac{d\mathbf{x}_p}{dt} = \mathbf{u}(\mathbf{x}_p), \quad \frac{d\boldsymbol{\omega}_p}{dt} = (\boldsymbol{\omega} \cdot \nabla) \mathbf{u}(\mathbf{x}_p) + \nu \Delta \boldsymbol{\omega}(\mathbf{x}_p) \quad (2)$$

194 while volumes remain constant due to the incompressibility. These convection–diffusion equations  
195 are solved in a viscous splitting algorithm, alternating convection and diffusion steps.

196 In the convection step, the particle velocities and strain needed in the right hand side of (2) are  
197 obtained as follows: the vorticity carried by the particles is first interpolated on a fixed cylindrical  
198 grid and velocity is evaluated by means of grid-based Poisson solvers. More precisely, the velocity  
199 is decomposed using the Helmholtz decomposition

$$\mathbf{u} = \bar{\mathbf{u}} + \nabla \times \boldsymbol{\psi} + \nabla \phi \quad (3)$$

201 where  $\bar{\mathbf{u}}$  is the irrotational field with the prescribed far field behavior.

202 The stream function  $\boldsymbol{\psi}$  satisfies in the fluid domain  $\Omega$  the system

$$-\Delta \boldsymbol{\psi} = \boldsymbol{\omega}, \quad \nabla \cdot \boldsymbol{\psi} = 0$$

204 and the potential  $\phi$  is used to enforce the no-through boundary condition  $\mathbf{u} \cdot \mathbf{n} = 0$ —where  $\mathbf{n}$   
205 denotes the outward unit normal vector—on the cylinder surface  $\Gamma$ :

$$\Delta \phi = 0 \quad \text{in } \Omega, \quad \frac{\partial \phi}{\partial n} = -(\nabla \times \boldsymbol{\psi}) \cdot \mathbf{n} \quad \text{on } \Gamma$$

207 Once velocity and its derivatives are computed on the grid, these quantities are interpolated back  
208 to particles. This allows next to push particles and update their circulations. A fourth order  
209 Runge–Kutta time-stepping is used to solve the underlying ordinary differential equations.

210 After each convection step, particles are remeshed on a regular cylindrical grid. The kernel used  
211 to interpolate vorticity from the particles to the grid and to remesh particles is a third order  
212 piecewise cubic spline.

213 Particle vorticities are finally redistributed to simulate diffusion. The no-slip boundary condi-  
 214 tion  $\mathbf{u} \cdot \boldsymbol{\tau} = 0$  (imposing a given slip can be done with straightforward modification) is satisfied in  
 215 the diffusion step through the flux on  $\Gamma$  of the tangential components of the vorticity: if  $u_\theta, u_z$  are  
 216 the tangential components of the velocity in the azimuthal and spanwise directions at the end of  
 217 an advection step, one solves for the corresponding components of the vorticity the following  
 218 boundary conditions on the surface of the cylinder

$$v \frac{\partial \omega_z}{\partial \mathbf{n}} = -\frac{u_\theta}{\Delta t}, \quad v \left( \frac{\omega_\theta}{r} + \frac{\partial \omega_\theta}{\partial \mathbf{n}} \right) = \frac{u_z}{\Delta t}$$

220 where  $v$  is the viscosity and  $\Delta t$  is the time-step. These vorticity boundary conditions are imple-  
 221 mented by means of integral equations.

222 To summarize, each time-step consists of the following sequence of operations:

- 223 • interpolate particle vorticity on a cylindrical grid,  
 224 • compute stream functions and potential on the grid,  
 225 • differentiate these quantities on the grid to obtain velocity and strain on the grid,  
 226 • interpolate velocity and strain back to particles,  
 227 • push particles and update vorticity,  
 228 • remesh particles on regular locations,  
 229 • compute the residual slip on the boundary and diffuse vorticity among particles with the appro-  
 230 priate vorticity fluxes to enforce the desired slip.

231 Several features of wake dynamics can be used to check the accuracy of the method. It is well-  
 232 known that when the viscosity is small enough (i.e.  $Re > 190$ ), two-dimensional solutions are  
 233 unstable and solutions become fully 3D. Fig. 3 illustrates the evolution of drag and lift and the  
 234 production of streamwise vorticity that goes together with the transition from 2D to 3D of the  
 235 wake for a Reynolds number of 300. For moderate Reynolds numbers the instabilities involved in  
 236 this transitions are mainly of two kinds, usually called *mode A* and *mode B* (see [22]). Refs. [15,16]  
 237 show that the present numerical method agrees well in all the classical diagnostics (drag and lift

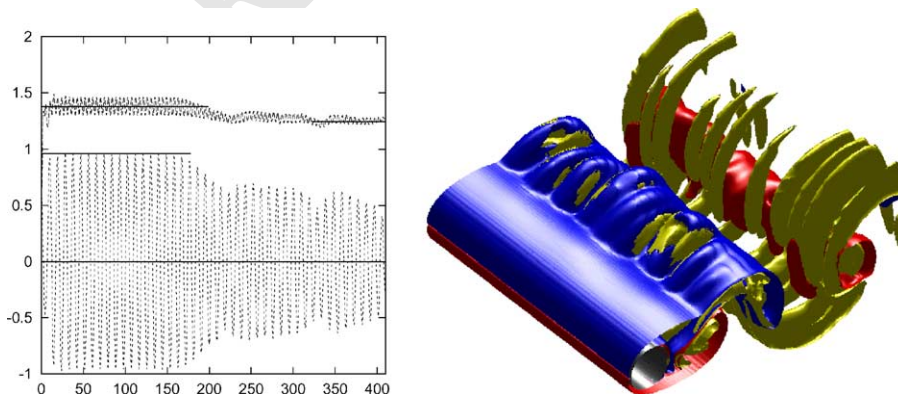


Fig. 3. Effect of three-dimensionality on drag and lift coefficients at  $Re = 300$  (left picture) and vorticity isovalues of post-transient three-dimensional flow at  $t = 270$  (right picture) (from [15]).



238 coefficients, Strouhal numbers, exponential growth of instabilities, energy spectrum, spectral  
 239 profiles of unstable modes, coherent structures) obtained by other numerical or experimental  
 240 techniques. A nice feature of the method is its combination of robustness and accuracy. Let us  
 241 point out that robustness was crucial to allow direct numerical simulations even in presence of  
 242 severe boundary manipulations, as those considered in the sequel.

### 243 3.2. Control strategy using rotation

244 As a first example of control for the cylinder wake, we consider the case of spanwise invariant  
 245 oscillations of the cylinder. It has been shown experimentally in [21] that a fast oscillating rotation  
 246 of the cylinder leads to a substantial drag reduction. This fact, already observed in 2D simulations  
 247 [6] has recently been confirmed in fully 3D simulations [15,16].

248 An additional property of this kind of control demonstrated in these references is the two-di-  
 249 mensionalization of the flow in a large neighborhood of the body when the angular velocity is  
 250 large enough. The dimension of the flow is measured by means of *directional enstrophies*: the  
 251 spanwise and orthogonal enstrophies

$$Z_z = \int_{\Omega} \omega_z^2 dv, \quad Z^\perp = \int_{\Omega} \omega_x^2 + \omega_y^2 dv \quad (4)$$

253 represent the amount of vorticity parallel or orthogonal to the cylinder axis. Global enstrophy is  
 254 recovered by  $Z = Z_z + Z^\perp$ .

255 When  $Z^\perp$  is small, the flow is mainly two-dimensional, while when  $Z^\perp$  and  $Z_z$  are close, the flow  
 256 is completely three-dimensional. The values of  $Z^\perp$  obtained at  $Re = 500$  for an oscillating rotation  
 257 (half a revolution of amplitude, once and twice the natural frequency of flow) are plotted in Fig. 5.

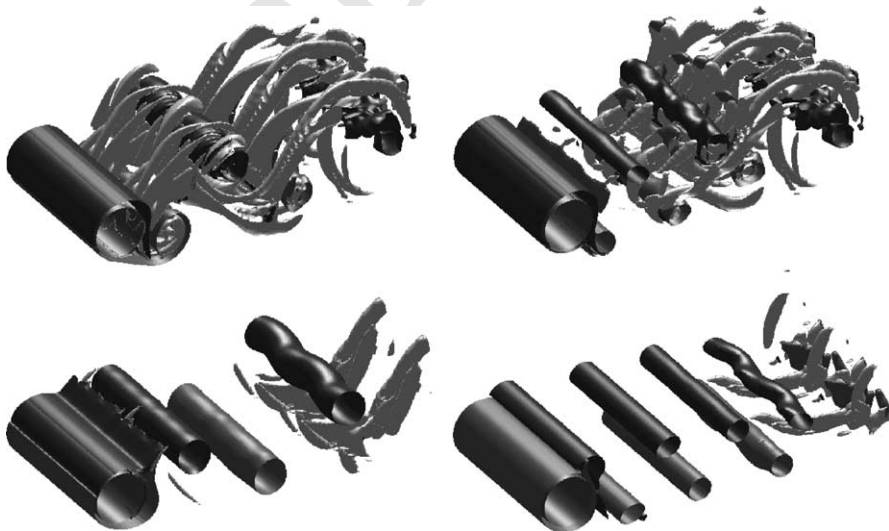


Fig. 4. Surfaces of isovorticity after activation of the rotation locked on the flow self-frequency (left pictures) and on twice this frequency (right pictures) (from [15]).

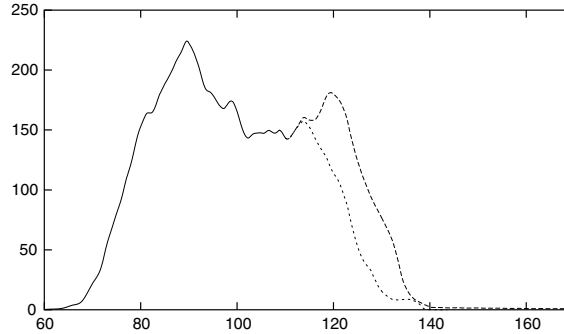


Fig. 5. Orthogonal enstrophy versus time for an oscillating rotation of the cylinder. —: rotation lock on the flow self-frequency, - - -: twice the flow self-frequency (from [15]).

258 Isosurfaces are plotted in Fig. 4 for a control rotation locked on once and twice the flow self-  
 259 frequency.

260 In these calculations and in the following, we used a resolution of  $256 \times 128 \times 128$  points in the  
 261 computational domain

$$(r, \theta, z) \in \Omega = [1, 1 + 4\pi] \times [-\pi, \pi] \times [0, 4\pi]$$

263 for a cylinder of radius unity. Periodicity was assumed in the spanwise direction.

### 264 3.3. Control strategy using tangential velocity profiles

265 In order to propose more effective control strategies, we now consider the case when cylinder  
 266 rotation is a function of  $\theta$  and  $z$ . Recent results for 2D cylinders [11] have actually allowed to find  
 267 optimal tangential velocity profiles distributed on 16 points evenly distributed around the cylin-  
 268 der. We chose, as a base profile, a function  $f$  that approximately fits these values and allowed  
 269 spanwise variation along modes A and B which describe the 3D instabilities in the wake. Our  
 270 function  $f$  has the following expression:

$$f(\theta) = -\sin\left(\frac{3 \cdot 2\theta^3}{3 + \theta^{10}}\right)$$

272 This function, plotted in Fig. 6, exhibits two extrema, which are chosen near the separation  
 273 points in order to reduce the drag: the profile  $f$  is a smooth function approaching values obtained  
 274 in [11]. This smooth function can be seen as a regularization of the piecewise constant function  $\tilde{f}$   
 275 plotted in Fig. 6, which could correspond experimentally to a 6-ribbons control.

276 To account for spanwise variations, the present computations use the four control parameters

$$\mathbf{C} = \begin{pmatrix} C_1 \\ C_2 \\ C_3 \\ C_4 \end{pmatrix} \quad (5)$$

278 and the azimuthal tangential velocity profile on the body is given by

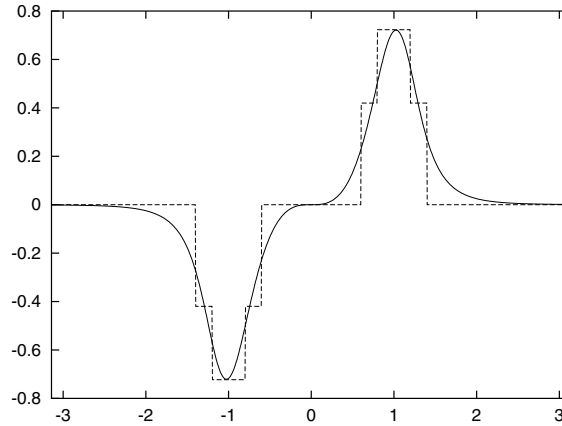


Fig. 6. Shape of functions  $f$  and  $\tilde{f}$ , extrema at  $\pm 0.723$  and mid-value at  $\pm 0.42$ .

$$V_{\text{slip}}(\theta, z) = \frac{D}{2} f(\theta) \mathbf{C} \cdot \begin{pmatrix} 1 \\ 2 \sin(2z/D) \\ 2 \sin(4z/D) \\ 2 \sin(8z/D) \end{pmatrix} \quad (6)$$

280 with  $-\pi \leq \theta \leq \pi$  and  $0 \leq z \leq 2\pi D$ . The control is started impulsively, at  $t_c = 270$  for 3D compu-  
 281 tations.

282 The energy involved in this control is

$$E = \frac{1}{2} \int_0^{2\pi D} \int_{-\pi}^{\pi} V_{\text{slip}}(\theta, z)^2 R d\theta dz = 2\pi R^3 \|\mathbf{C}\|_2^2 \int_{-\pi}^{\pi} f(\theta)^2 d\theta \quad (7)$$

284 because cross terms have null mean value. The energy required in a particular control within this  
 285 class is thus given by the Euclidean norm of  $\mathbf{C}$ .

### 286 3.4. Two-dimensional control on 2D and 3D wakes

287 By two-dimensional control, we mean a velocity profile that does not depend on spanwise  
 288 coordinate, and thus can be applied to both 2D and 3D flows.

289 This implies that the control parameter set is  $\mathbf{C} = (C_1, 0, 0, 0)$ . Fig. 7 shows that the mean drag  
 290 coefficient, at  $Re = 300$ , drops from 1.38 down to 0.64—a 53% reduction—when  $C_1 = 1$ . The  
 291 effect of such a control on the vorticity field is plotted in Fig. 8, at different time from  $t_c = 270$  to  
 292  $t_c + 50 = 320$ . One may observe that, as for the case of constant rotation, the flow returns to a 2D  
 293 state, and that a substantial stretching of the recirculation zone occurs.

### 294 3.5. Control with 3D vorticity profiles

295 In order to first analyze individually the effects on the wake of various 3D forcings, we consider  
 296 the following control vectors:

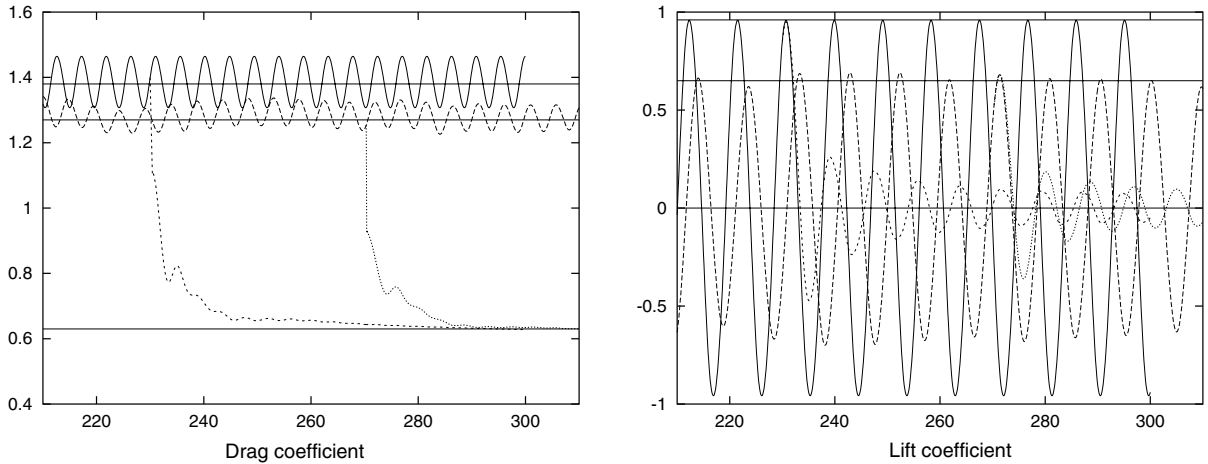


Fig. 7. Effect of 2D control ( $C_1 = 1$ ) on drag and lift coefficients: —: 2D without control, ---: 3D without control, ···: 2D with control, -·-·: 3D with control.

- 297 •  $\mathbf{C} = (0, 1, 0, 0)$ , with wavelength  $\lambda/D = \pi$ , matching the mode A instability wavelength at  $Re =$
- 299  $Re = 300$  (called *type A control* in the sequel),
- 300 •  $\mathbf{C} = (0, 0, 1, 0)$ , with wavelength  $\lambda/D = \pi/2$  intermediate between modes A and B wavelength
- 301 (*type A–B control*),
- 302 •  $\mathbf{C} = (0, 0, 0, 1)$ , with wavelength  $\lambda/D = \pi/4$ , matching the mode B instability wavelength (*type*
- 303  $\mathbf{C} = (1, 0, 0, 0)$ , spanwise invariant (*2D-type control*).

304 All these controls involve the same energy. The case  $\mathbf{C} = (1, 0, 0, 0)$  has already been discussed. As

305 an example, type A–B control points are shown in Fig. 9.

306 Fig. 10 contains snapshots of the vorticity field for two different cases. It shows to which extent

307 the forcing affects the topology of the wake.

308 Finally, combination of modes are considered as control functions. We focus our attention on

309 combination of 2D-type and A–B type controls, involving the same energy as in the last section.

310 This leads to the parameters

$$\mathbf{C} = \frac{1}{\sqrt{5}}(2, 1, 0, 0) \quad (8)$$

312 which means that the velocity has the following expression:

$$V_{\text{slip}}(\theta, z) = \frac{D}{\sqrt{5}}(1 + \sin(2z/D))f(\theta) \quad (9)$$

314 This profile, whose wavelength is  $\lambda/D = \pi/2$ , has a very special property. It breaks the von

315 Kármán streets, into streamwise structures of vorticity which find a quasi-stationary state, thus

316 preventing the shedding of the flow. To measure the amount of shedding, one can consider the

317 time evolution of the *mean spanwise vorticity*

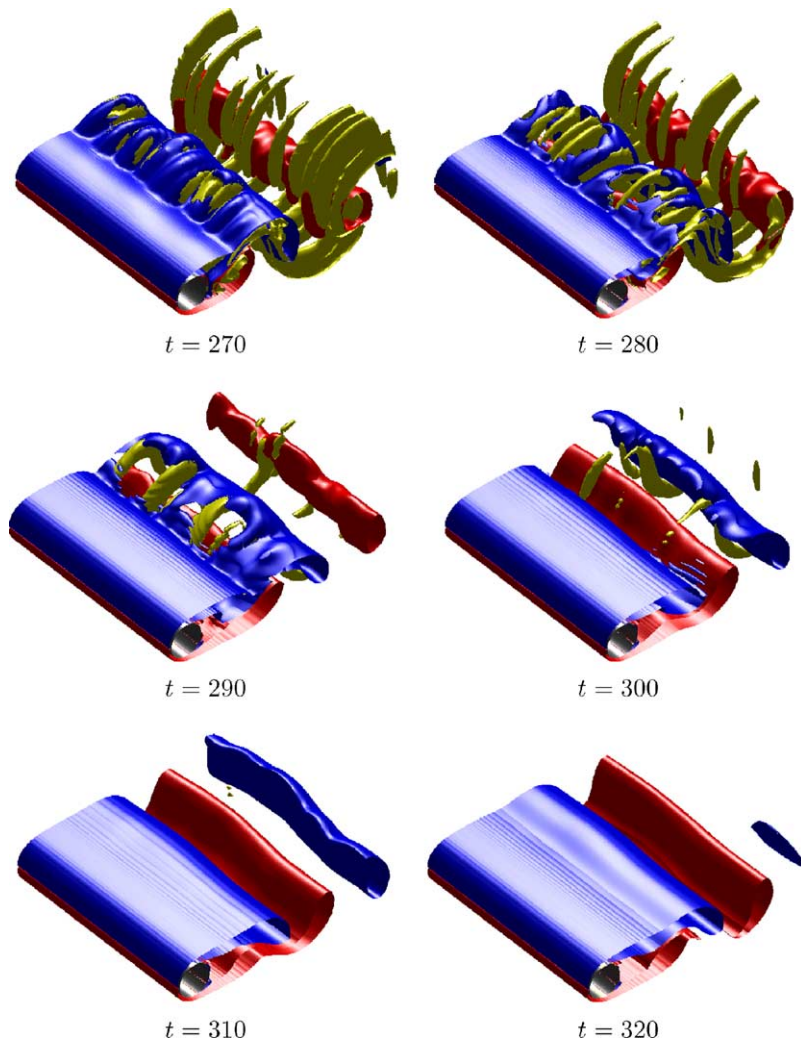


Fig. 8. Effect of the two-dimensional control  $C_1 = 1$ : vorticity field for a sequence of times.

$$\Gamma_z = \int_{\Omega} \omega_z \, dv.$$

319 When von Kármán alleys leave the computational domain (i.e. in presence of shedding), this  
 320 quantity oscillates. A lack of oscillation, as seen in Fig. 11, thus shows an absence of shedding.  
 321 Fig. 13 shows the vorticity isosurface corresponding to this particular type of wake.

322 Drag curves corresponding to an impulsively started control on either a 2D or a fully developed  
 323 3D wake, are plotted in Fig. 12. A comparison of this result with Fig. 7 suggests that the sup-  
 324 pression of shedding, resulting from a 3D forcing, induces an additional drag reduction of about  
 325 10% over a purely 2D control. This preliminary result brings some confidence that a fully 3D  
 326 optimization might improve 2D results.

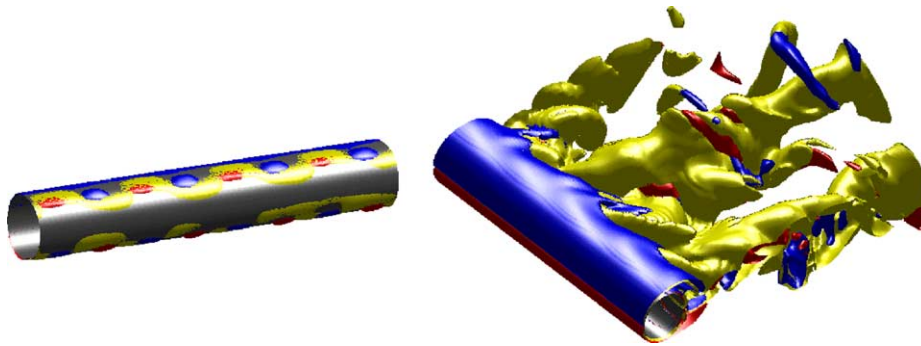
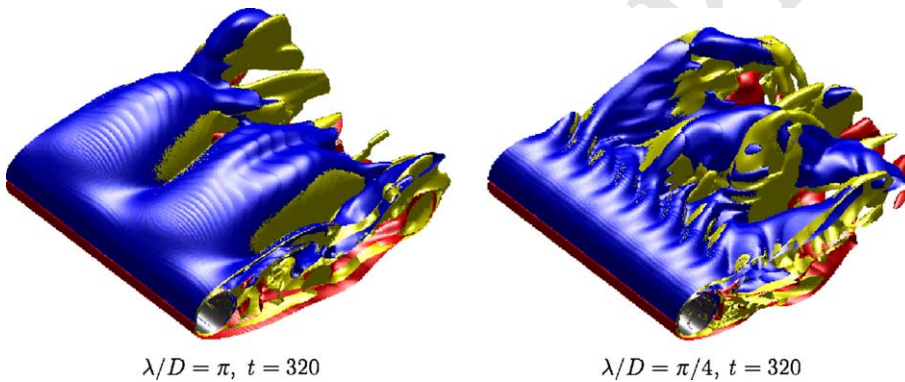


Fig. 9. Type A-B control: control points (left picture) and resulting flow at  $t = 380 = t_c + 110$  (right picture).



$\lambda/D = \pi, t = 320$

$\lambda/D = \pi/4, t = 320$

Fig. 10. Influence of control wavelength on the vorticity field.

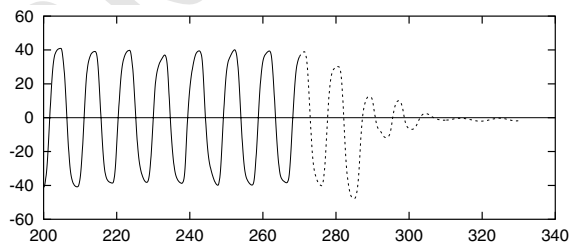


Fig. 11. Mean spanwise vorticity  $\Gamma_z$  versus time, using mixed control (type A-B and 2D) on a 3D flow. —: without control, ---: control on 3D flow.

327 **4. Conclusion and outlook**

328 We have considered two examples of wake optimization: the destruction of trailing vortices and  
329 the reduction of drag in a cylinder wake. The first example was simple enough, from the com-

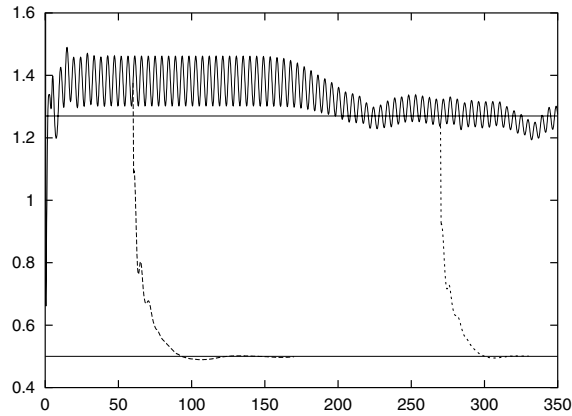


Fig. 12. Drag coefficient versus time, using mixed control (type A-B and 2D) on 2D and 3D flows. —: without control, ---: control on 2D flow, -.-: control on 3D flow. Constant values at 0.5, 1.27 and 1.38.

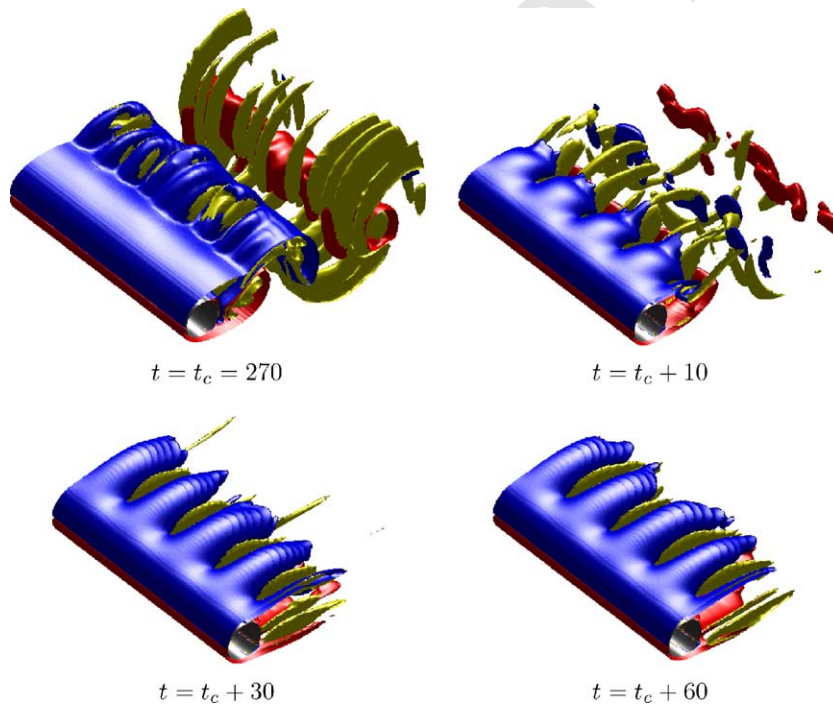


Fig. 13. Isosurface of vorticity, using mixed control applied on 3D flow.

330 putational point of view, to allow reasonably fast optimization by stochastic optimization. Fur-  
 331 thermore, this class of optimization techniques proved to be particularly flexible and opens the  
 332 possibility to incorporate more closely design constraints. In the second case, the complexity of  
 333 the flow dynamics makes a brute force stochastic optimization far too expensive. A reduction of  
 334 the number of parameters is crucial. Open loop control emphasizing the role of types A and B

335 instabilities give promising results which should open the way to efficient implementations of fully  
336 3D optimization algorithms.

## 337 5. Uncited reference

338 [1]

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